

Modeling JSC-1A Simulant Flow and Heat Transfer for the Helium Extraction & Acquisition Testbed



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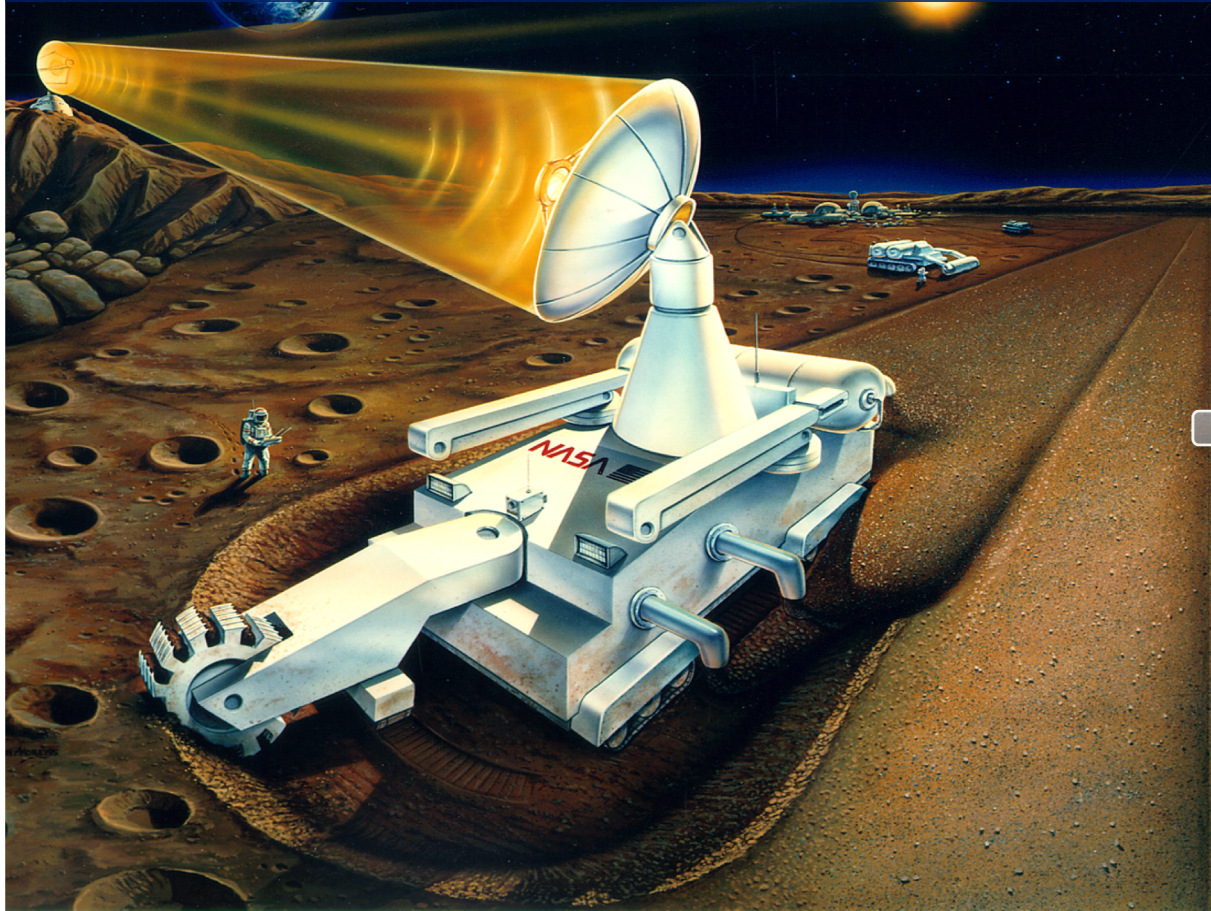
Outline

- Background: Research at the Fusion Technology Institute
- Overview of the Helium Extraction and Acquisition Testbed (HEAT)
- Granular Flow and Heat Transfer Modeling Approach
- Modeling Results
- Summary and Conclusion

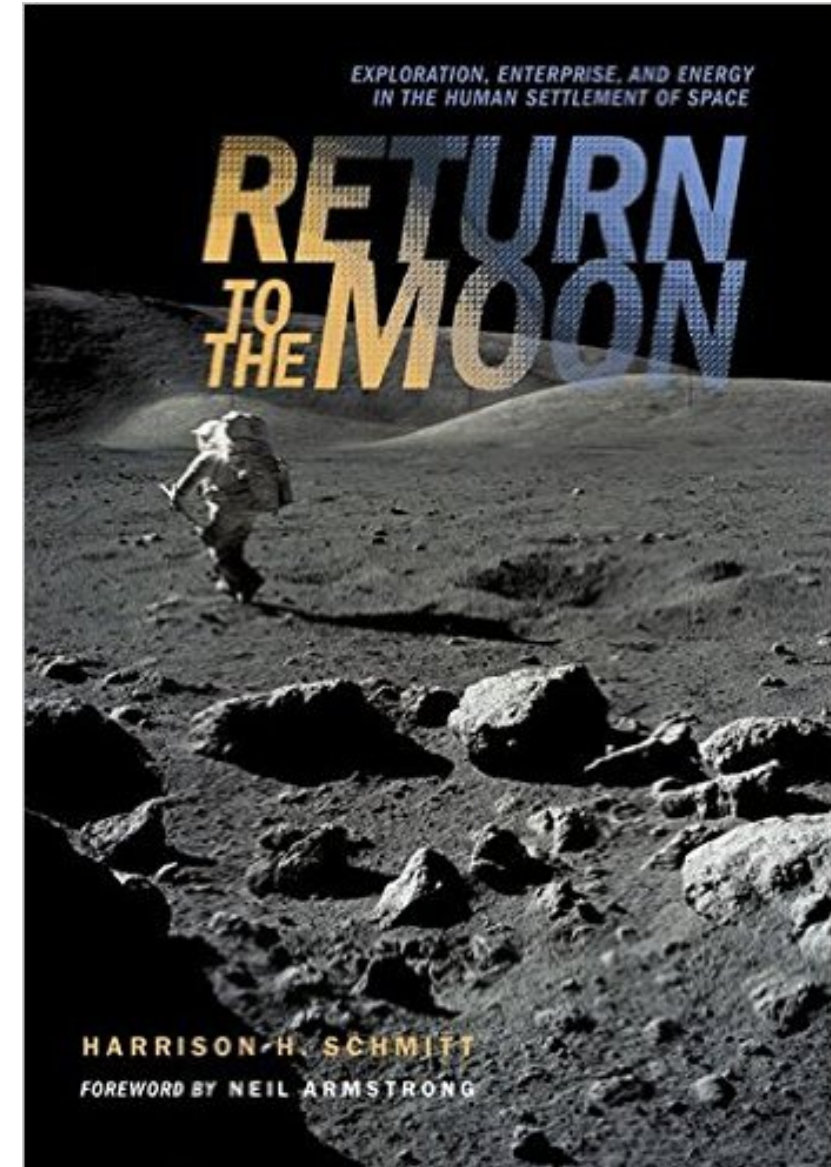
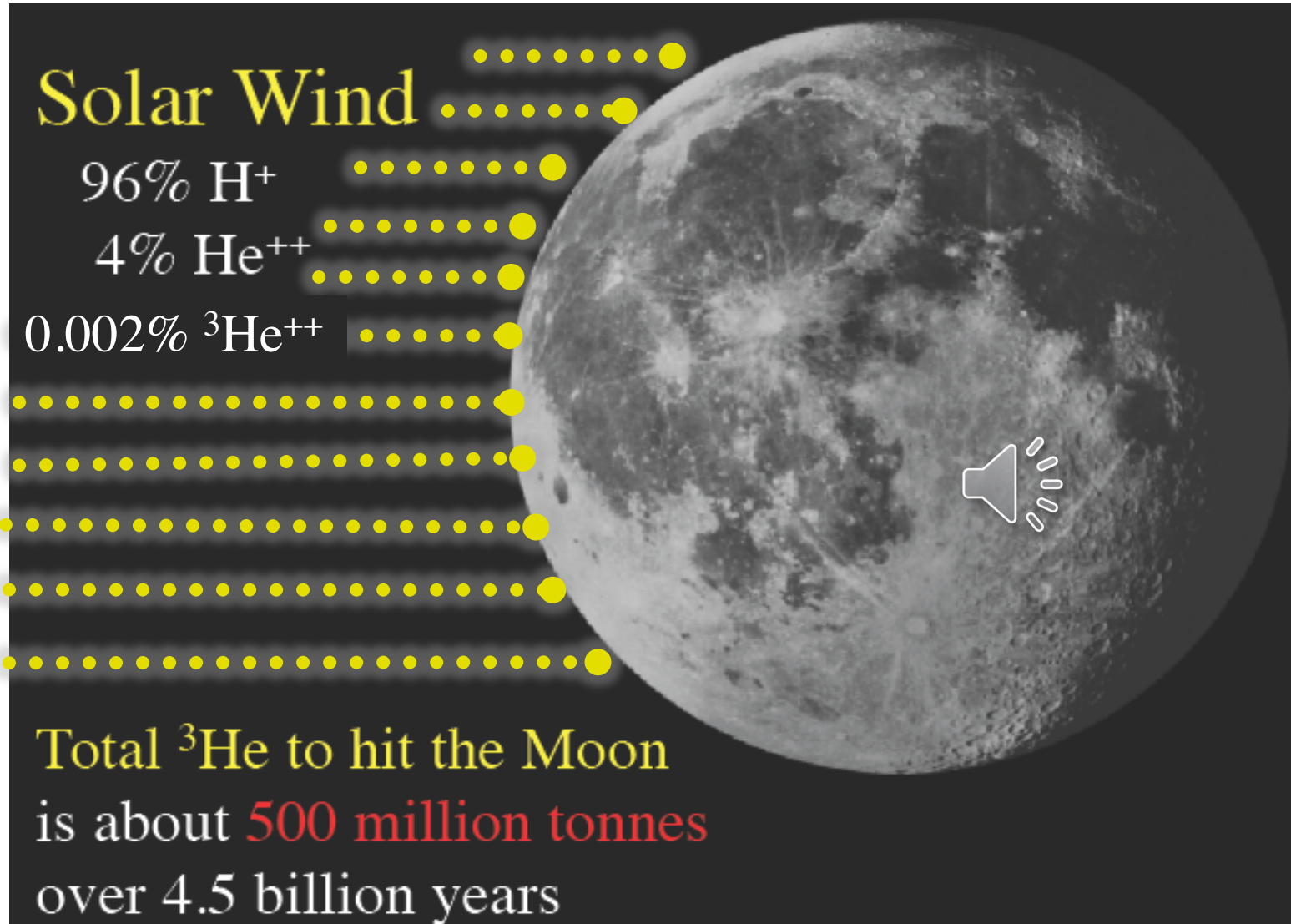
The FTI First Proposed Using Lunar Helium-3 For Fusion

Helium-3 & Lunar Volatiles for Fuel & Life Support

Schmitt and Olson, 2013

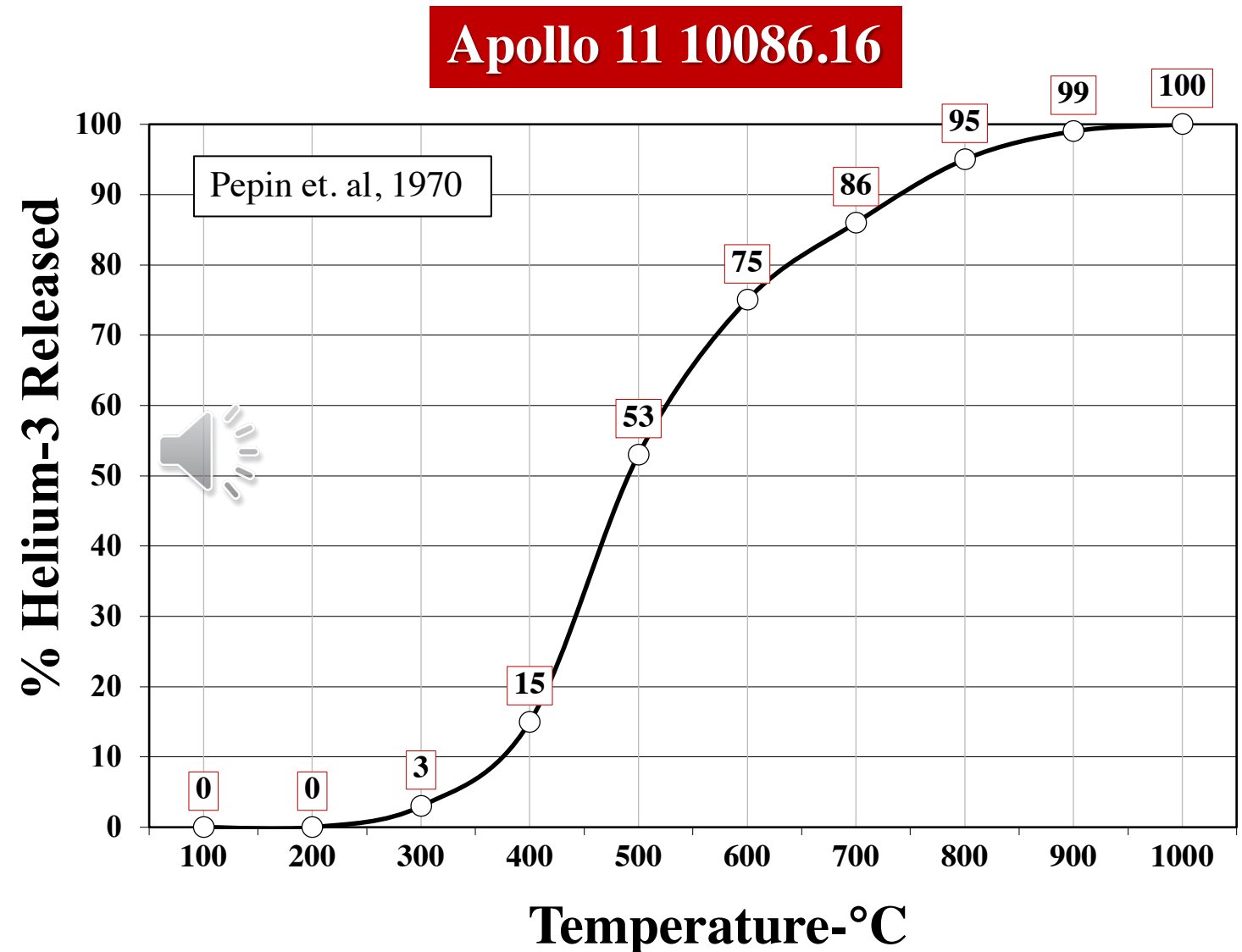


The Moon has retained over 1 million tonnes of ^3He



Heating Regolith Releases ^3He

- Heat to 700 °C to release 86% of embedded ^3He
- Peak release rate \sim 500 °C
- Agitation release – not yet quantified



There has been He-3 Miner Design Work at the FTI Since 1988

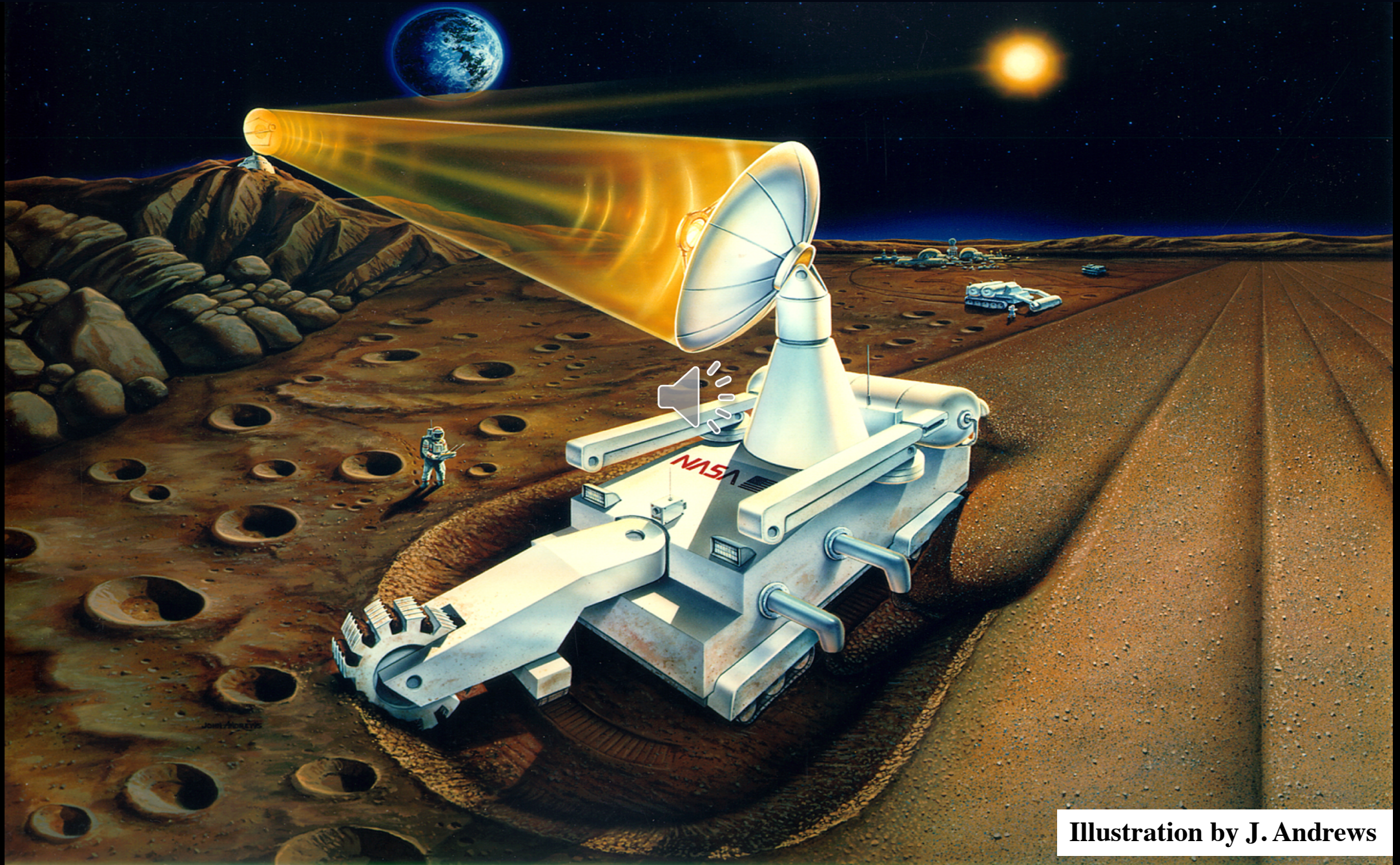
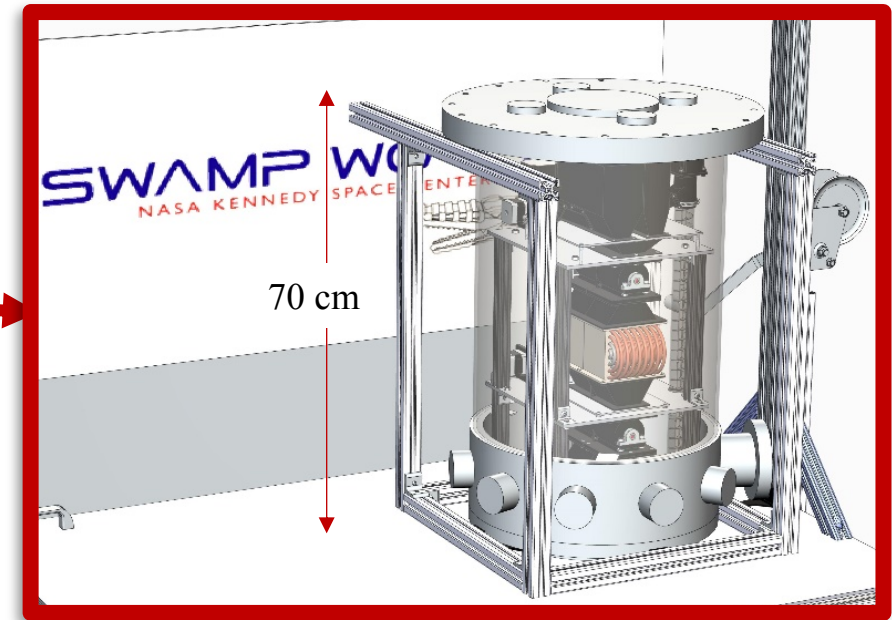
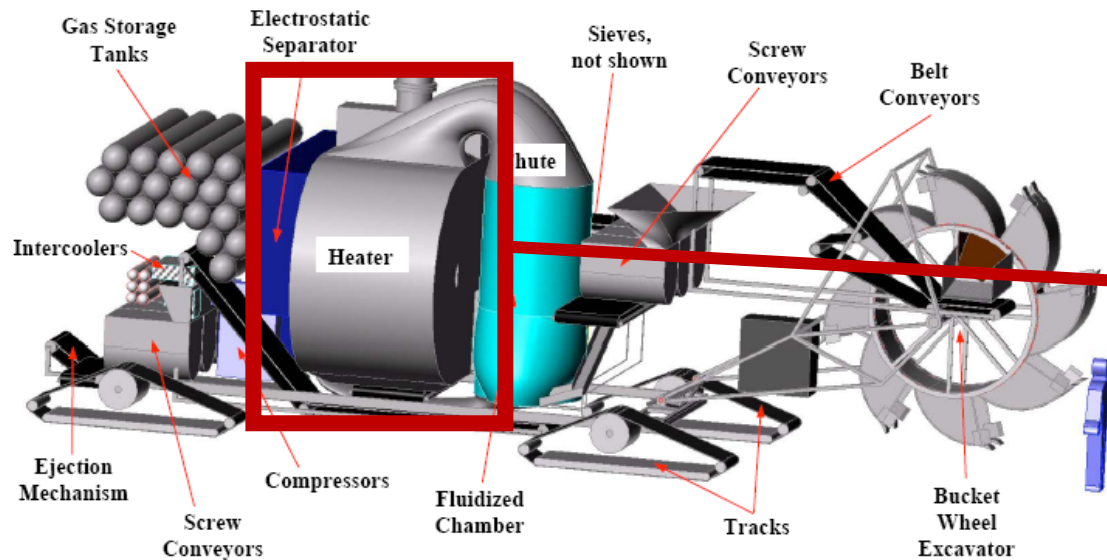
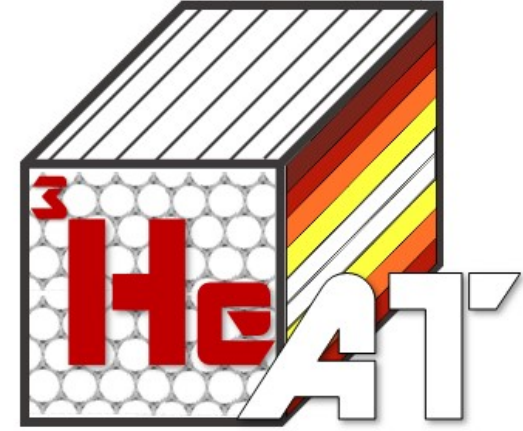


Illustration by J. Andrews

Research is Ongoing to Test a Scalable Version of the Volatiles Extraction System

Helium Extraction & Acquisition Test bed (HEAT)

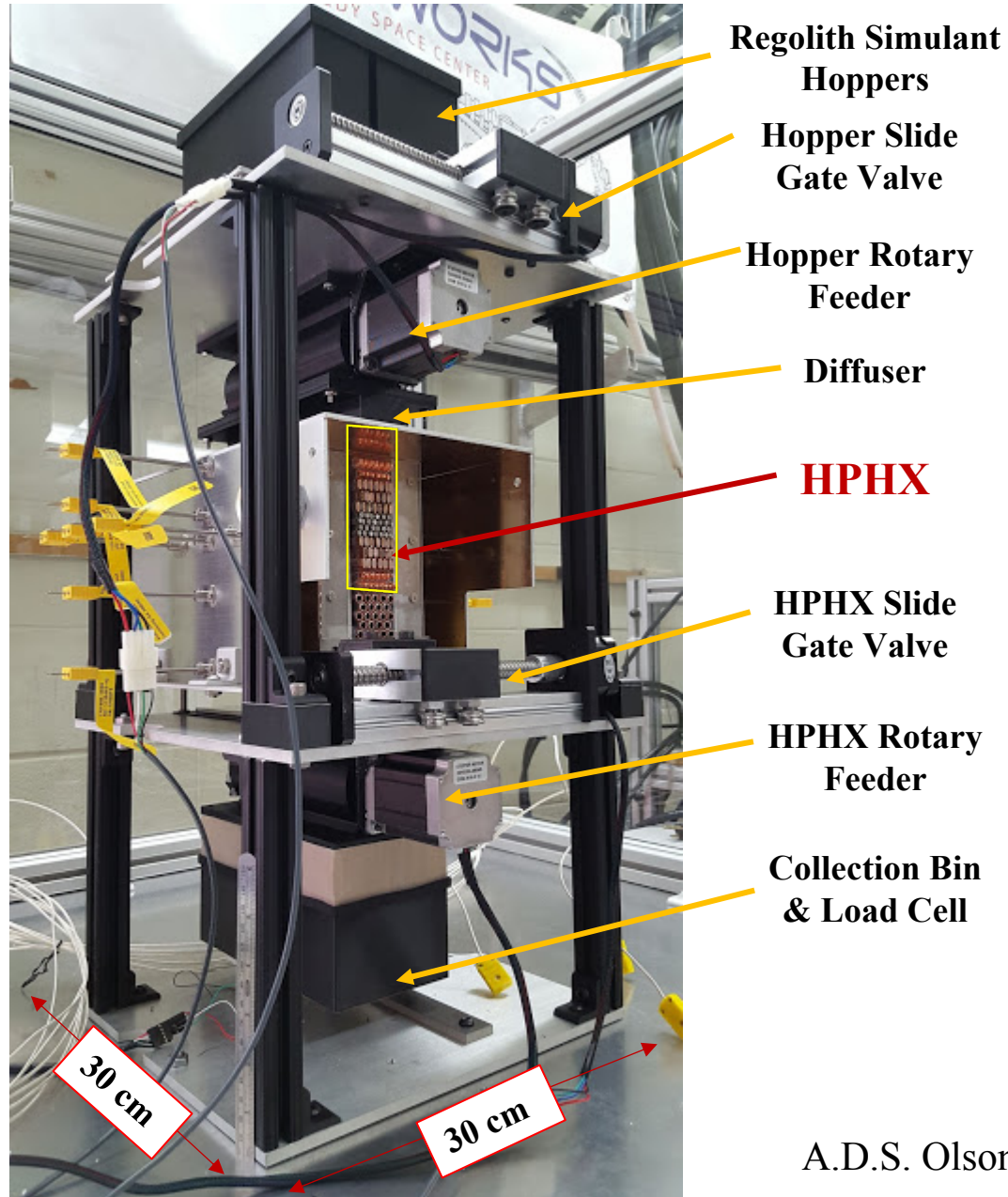
- Testbed for:
 - Thermal recuperation with a heat pipe heat exchanger
 - Volatiles extraction in a heat exchanger
- Laboratory scale (Technology Readiness Level – TRL 4)



Mark 3 Lunar Miner, Credit: M. Gajda



HEAT Can Test Volatile Extraction in a Heat Pipe Exchanger



- Heat Pipe Heat Exchanger (HPHX)
- Maximum mass flow rate (0.62 kg/s) (1:250 scale of Mark II)
- Heats regolith from $\sim 20^{\circ}\text{C}$ up to 450°C to release 30-50% of embedded helium
- Design for $<100\ \mu\text{m}$ JSC-1A regolith simulant
- Instrumentation
 - IR Imager, Thermocouples (K), Load Cell & RGA Instrumentation

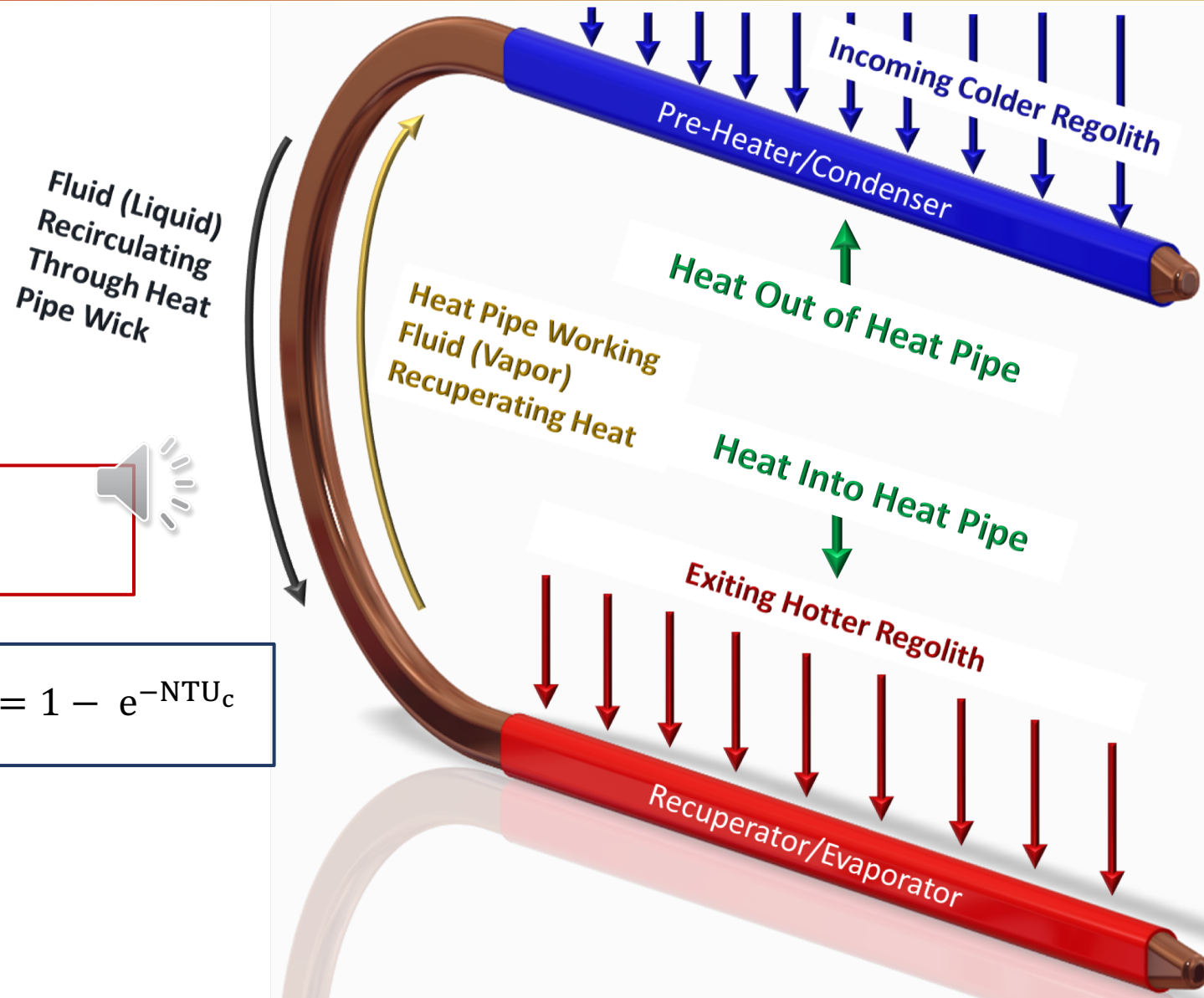
The Heat Pipe Analysis is Based on a Counterflow HX Model

- Energy balance of cold & hot regolith flow
- Effectiveness – NTU method
- Heat pipe effectiveness - function of thermal conductance regolith flow capacitance rate
- Thermal conductance - product of heat transfer coefficient and surface area

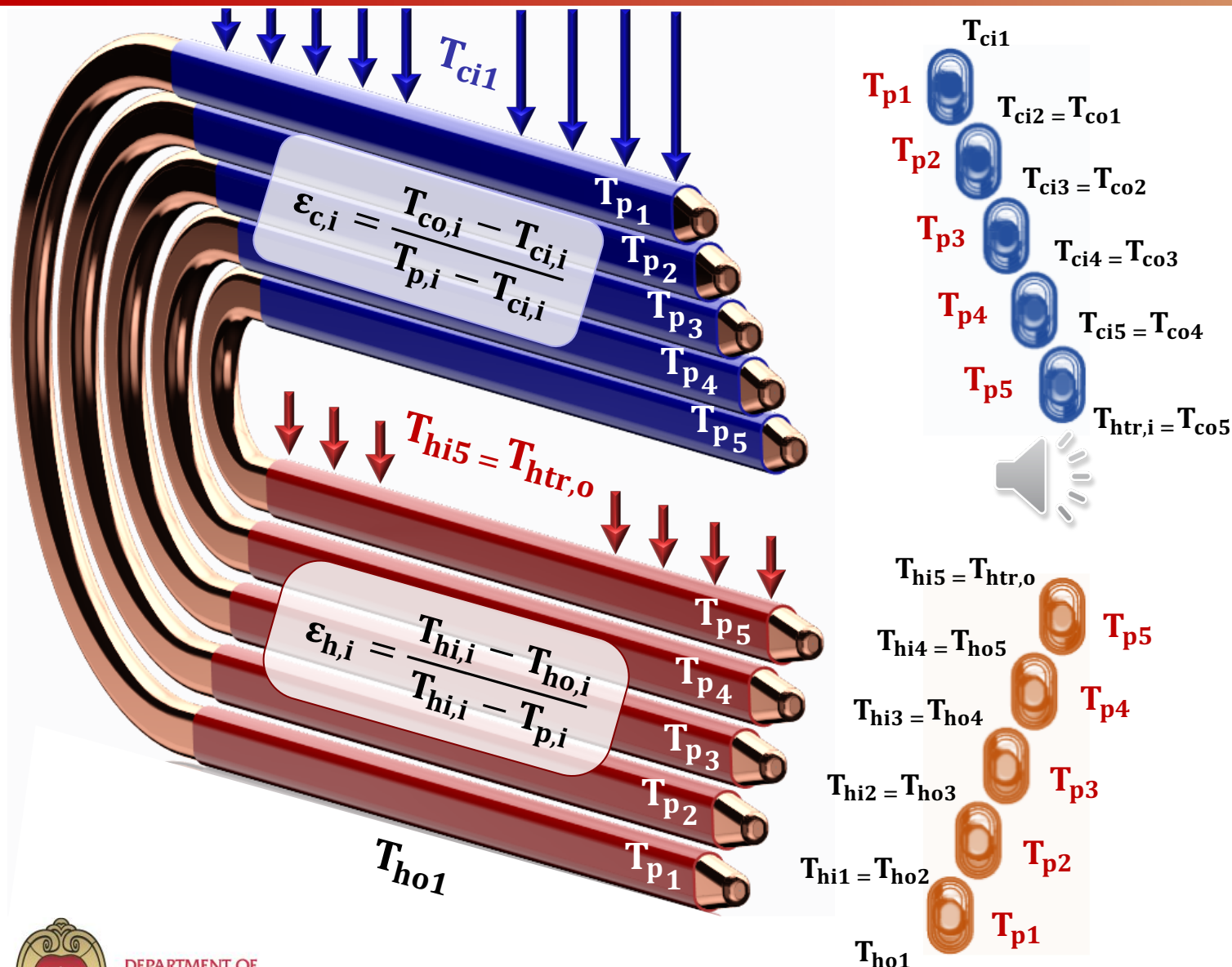
$$Q = C_h(T_{hi} - T_{ho}) = C_h \varepsilon_h (T_{hi} - T_p) \\ = C_c(T_{co} - T_{ci}) = C_c \varepsilon_c (T_p - T_{ci})$$

$$\varepsilon_h = \frac{T_{hi} - T_{ho}}{T_{hi} - T_p} = 1 - e^{-NTU_h} \quad \varepsilon_c = \frac{T_{co} - T_{ci}}{T_p - T_{ci}} = 1 - e^{-NTU_c}$$

$$NTU_h = \frac{h_h A_h}{\dot{C}_h} \quad NTU_c = \frac{h_c A_c}{\dot{C}_c}$$



The Heat Pipe HX is Modeled like a Staged Counterflow HX



System of equations solved in EES



Key Inputs

- Regolith and pipe friction and thermal properties
- Heat transfer coefficient functions
- Regolith inlet temperature (T_{ci1})
- Regolith maximum temperature (T_{hiN})

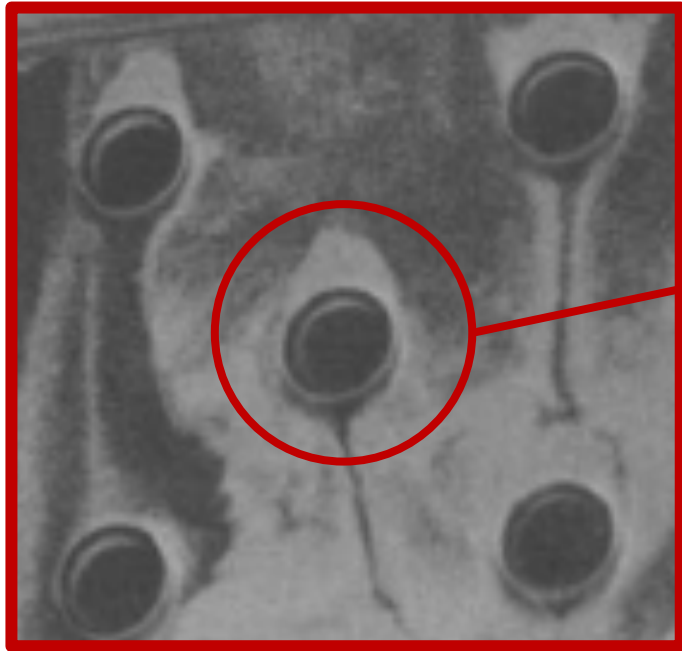
Outputs

- Heat pipe stage temperatures (N)
- Heat pipe heat transfer
- Regolith temperature vs. heat pipe stage
- Heater section inlet temperature
- Heater power and heat flux requirements

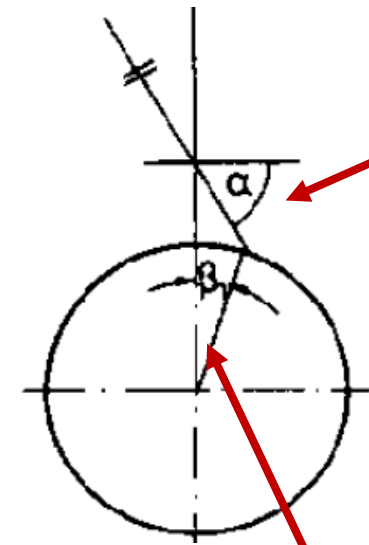
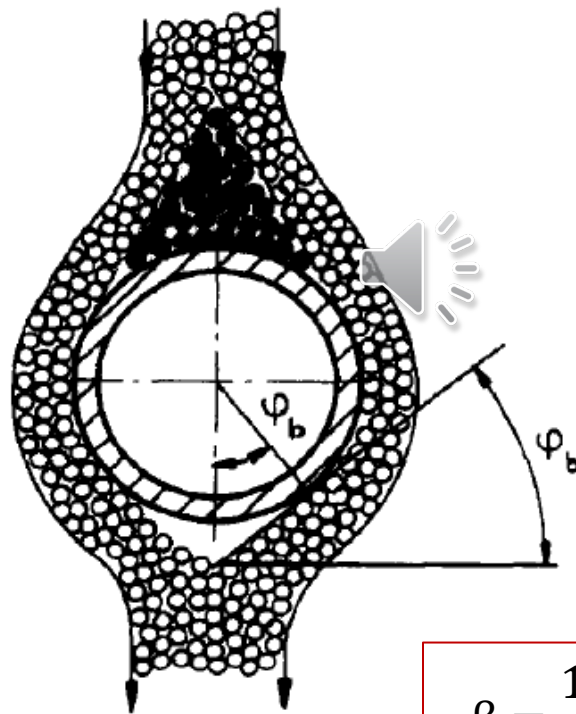


This Research Effort Focuses on an Analytical Flow Model

- The granular friction properties influence the flow channel shape
- The Niegsch model (Niegsch et al., 1994) incorporates the stagnation and void areas of flow



Credit: Niegsch et al.,1994

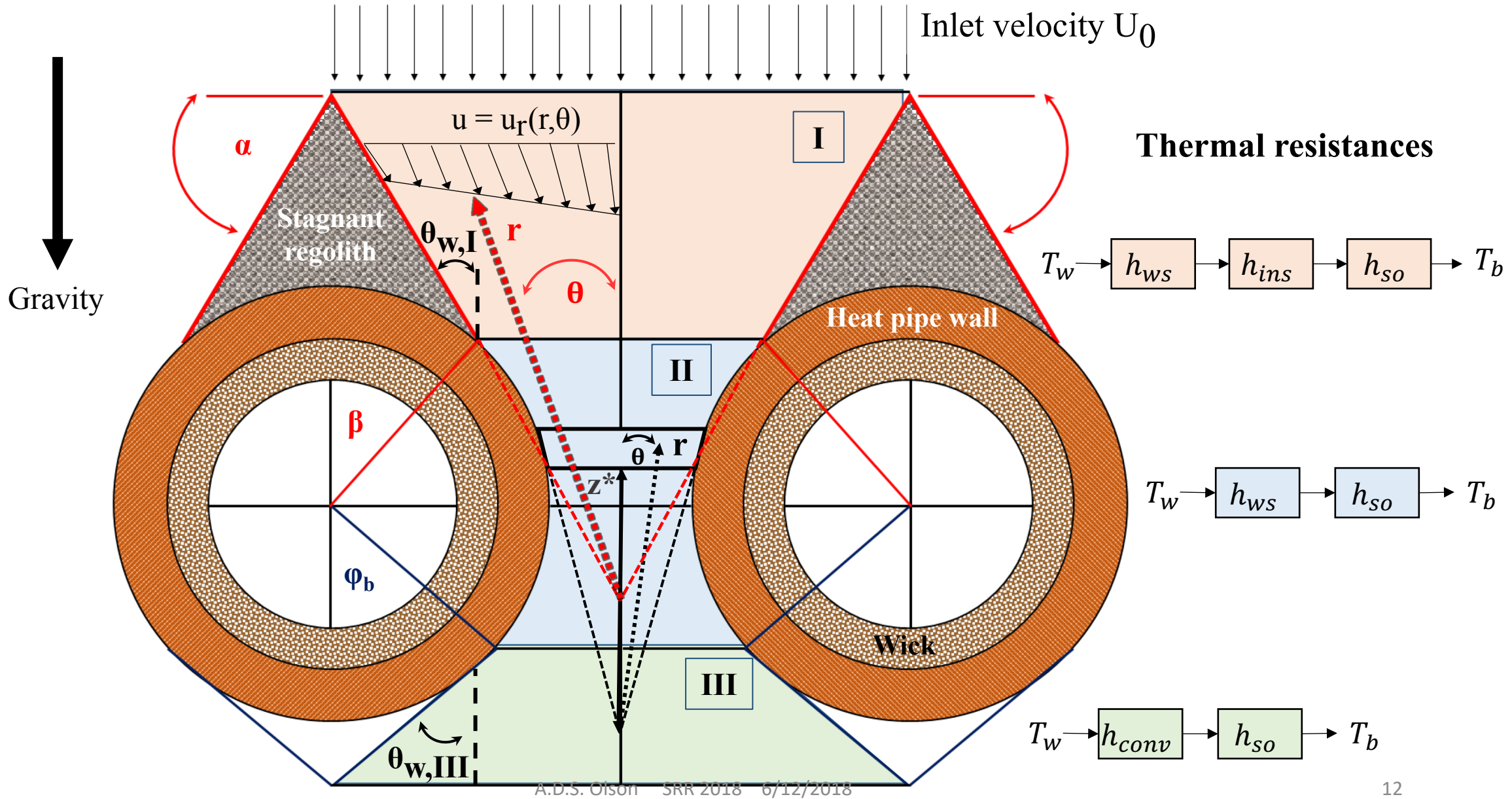


$$\alpha = \frac{\pi}{4} + \varphi_e$$

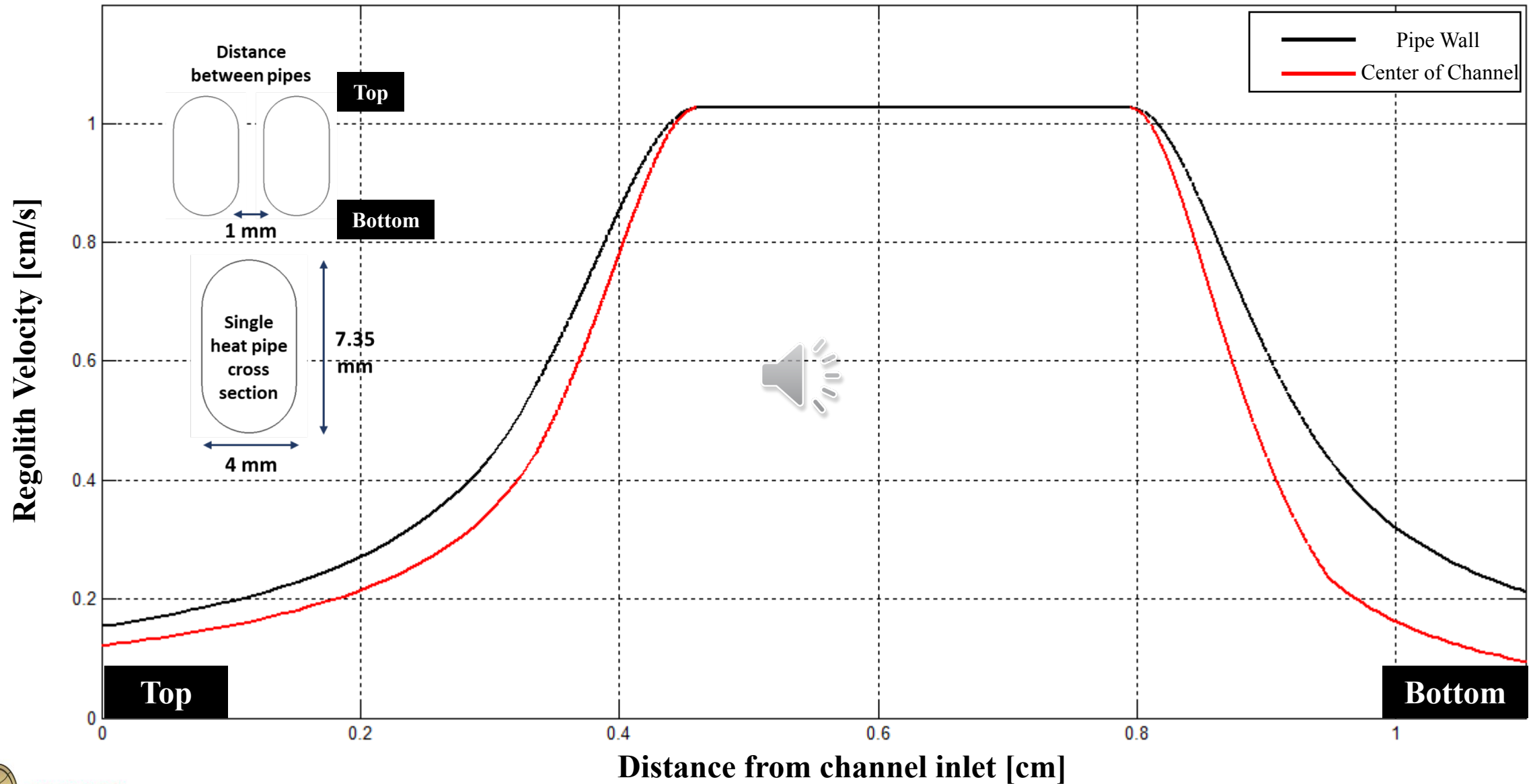
$$\beta = \frac{1}{2} \left[\cos^{-1} \left(\frac{1 - \sin(\varphi_e)}{2 \sin(\varphi_e)} \right) + \sin^{-1} \left(\frac{\sin(\varphi_w)}{\sin(\varphi_e)} \right) + \varphi_w \right]$$



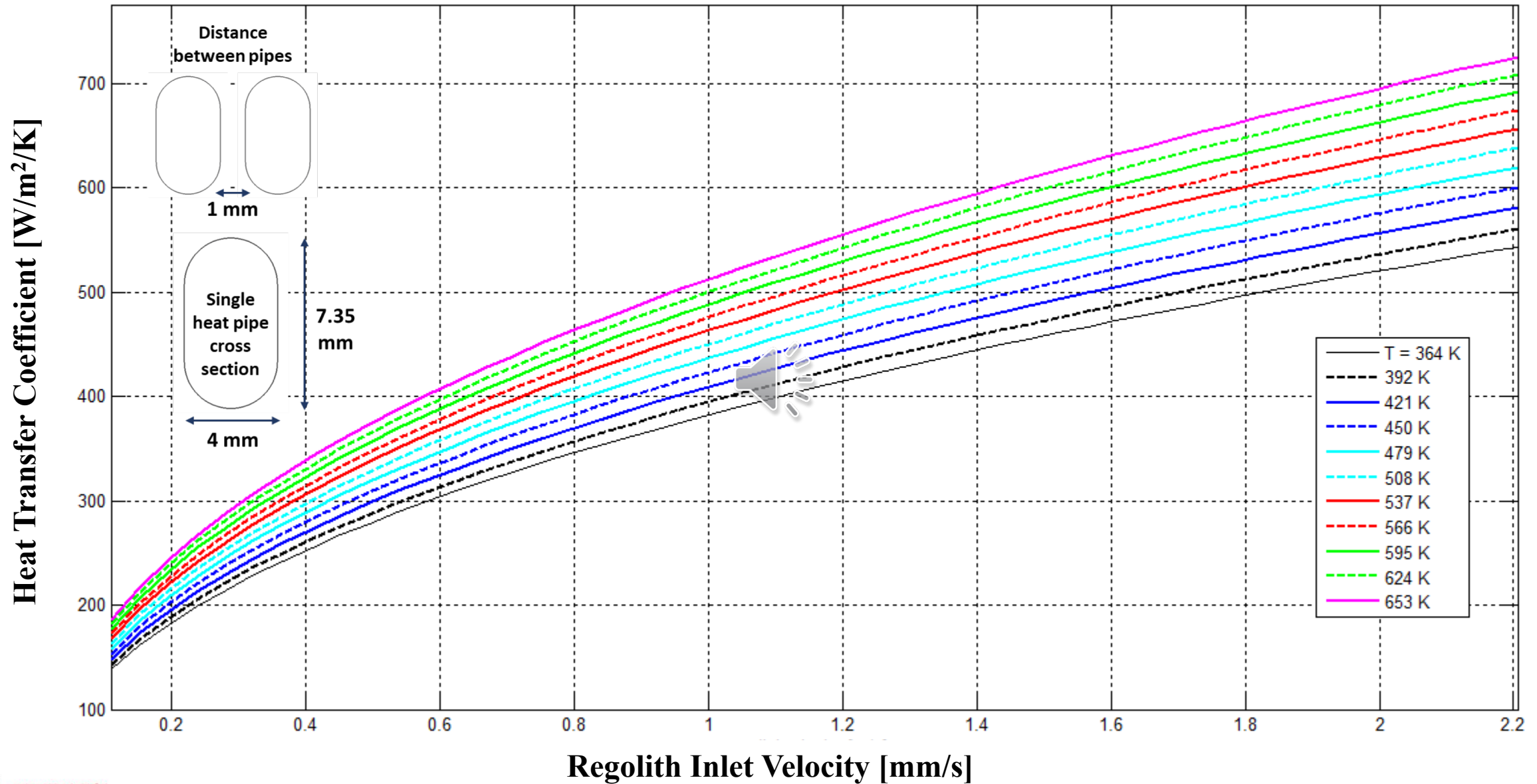
The Flow Model Produces A Velocity Field & Surface Heat Transfer Coefficient



Regolith Velocity vs. Position Between Heat Pipes

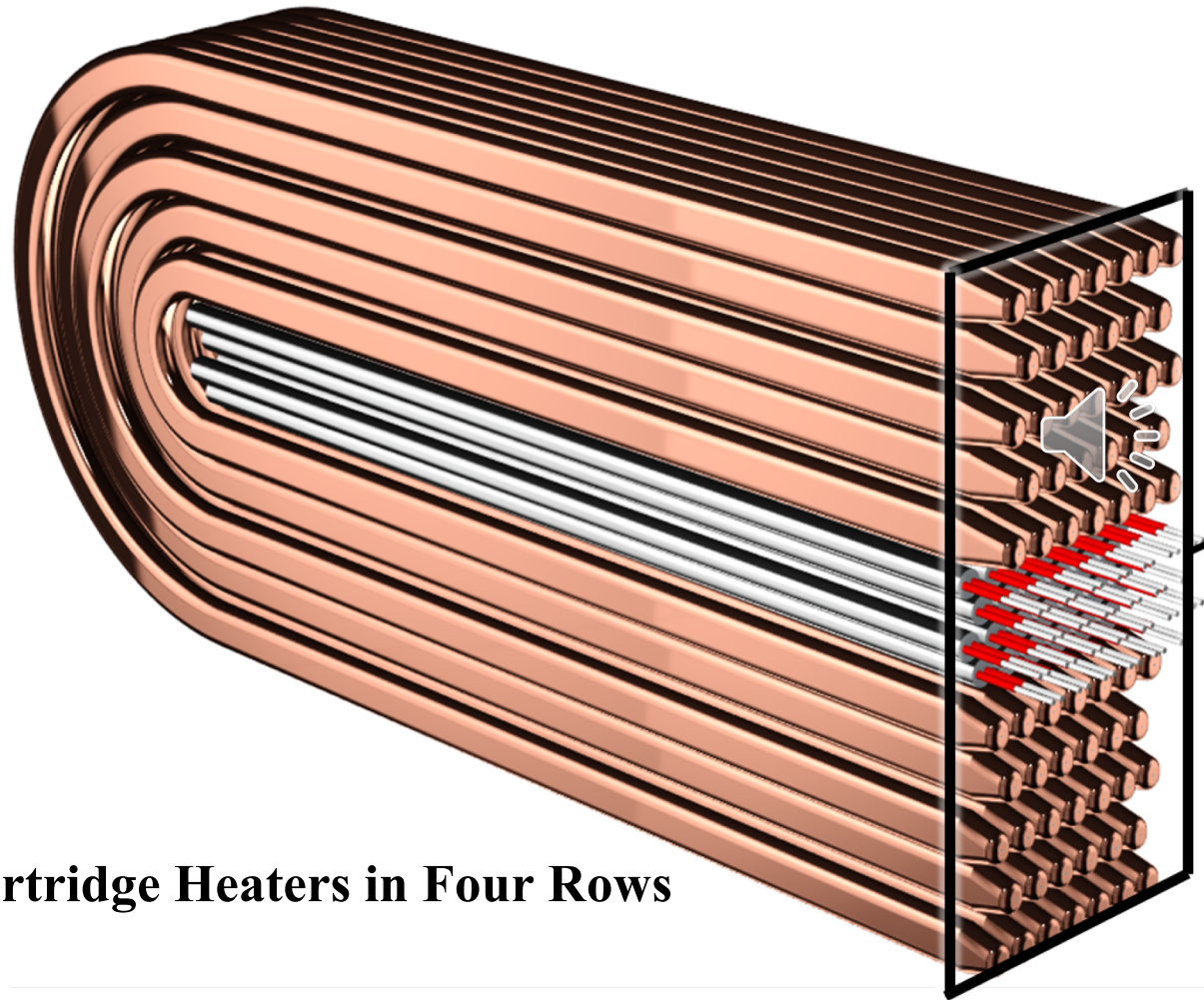


JSC-1A to Heat Pipe Heat Transfer Coeff. vs. Velocity and Regolith Temp.

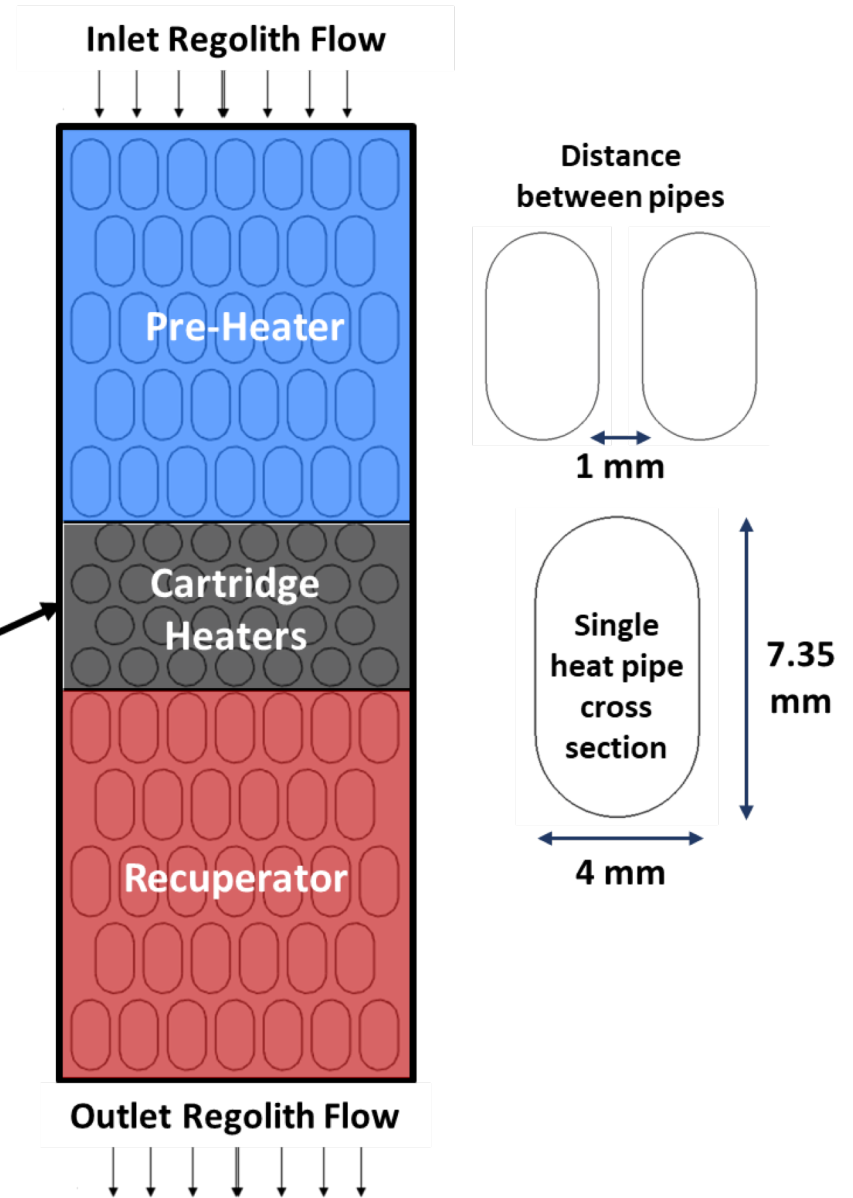


The HPHX Nominal Design Uses Five Stages of Flattened Heat Pipes

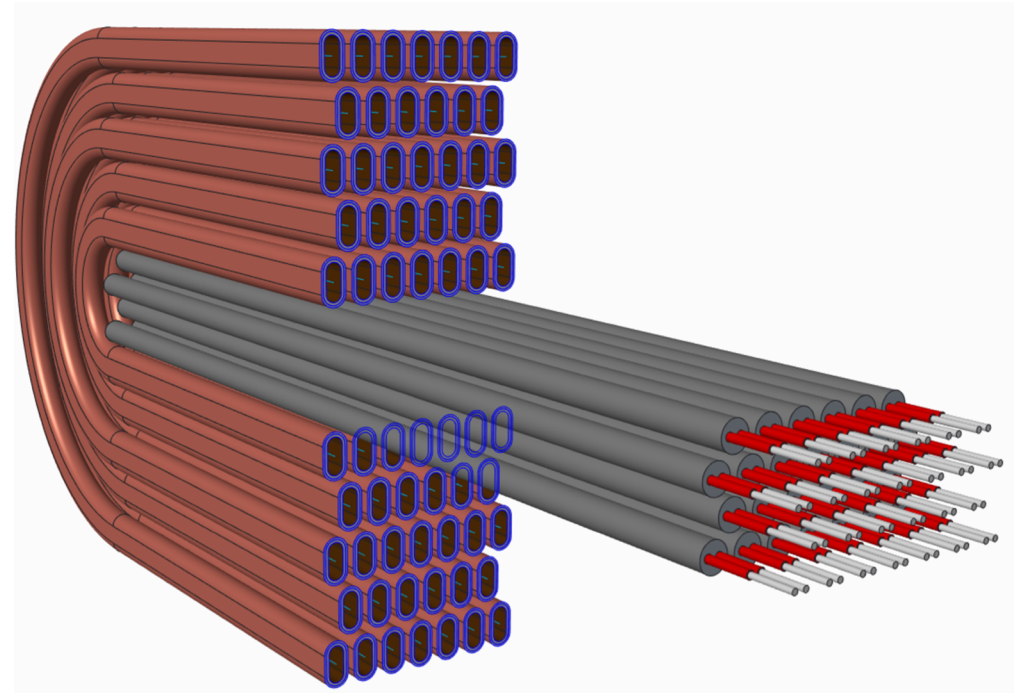
33 Heat Pipes in Five U-shaped stages



26 Cartridge Heaters in Four Rows

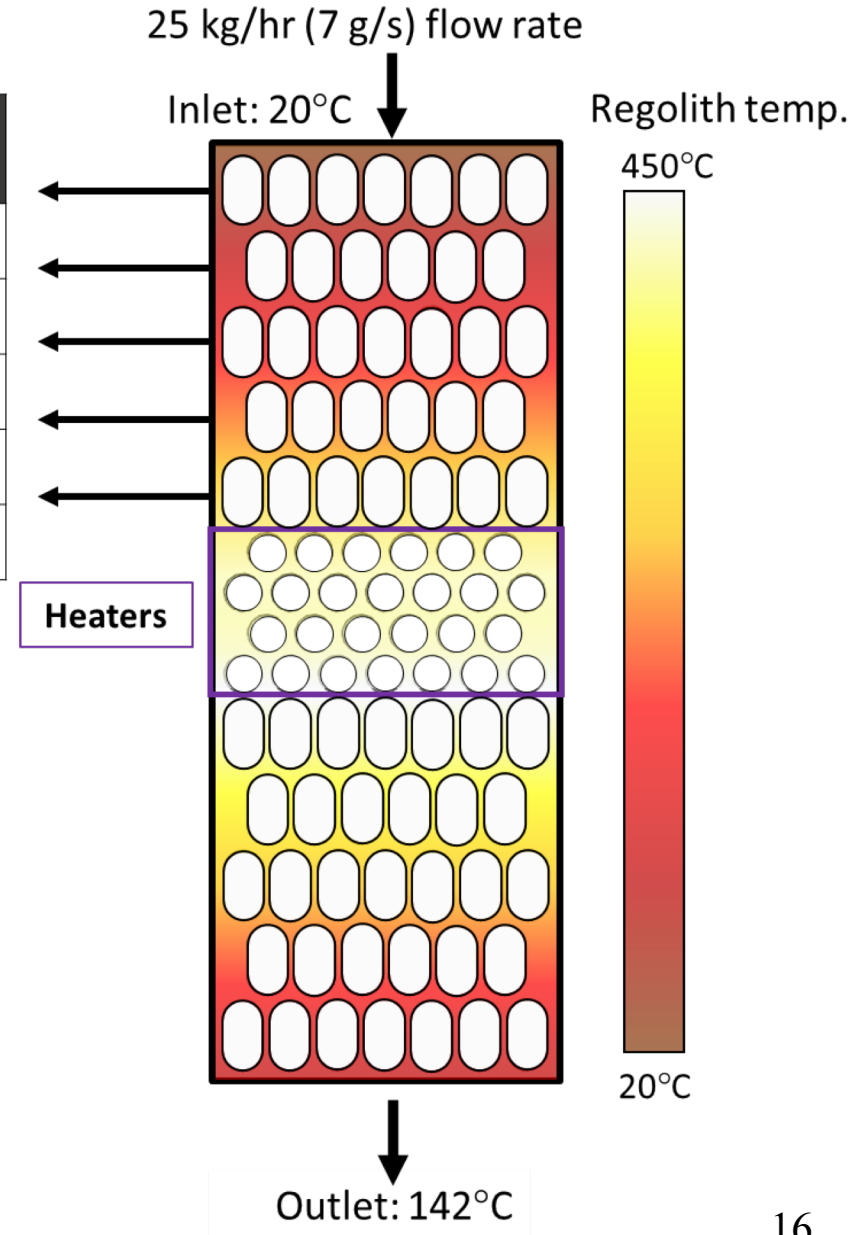


HPHX Predicted to Recuperate 72% of JSC-1A Thermal Energy



Fluid	HP Temp.	HP Heat	Stage
Water	101°C	34 W	1
Water	147°C	54 W	2
Dowtherm A	207°C	66 W	3
Dowtherm A	275°C	80 W	4
Dowtherm A	349°C	86 W	5

Input Power	922 W
Recoup. Efficiency	72 %
Cartridge Power	42 W
Heater Heat Flux	2.2 W/cm ²
Heater Surf. Temp.	460 °C
Total Residence	78 seconds
Heater Residence	14 seconds

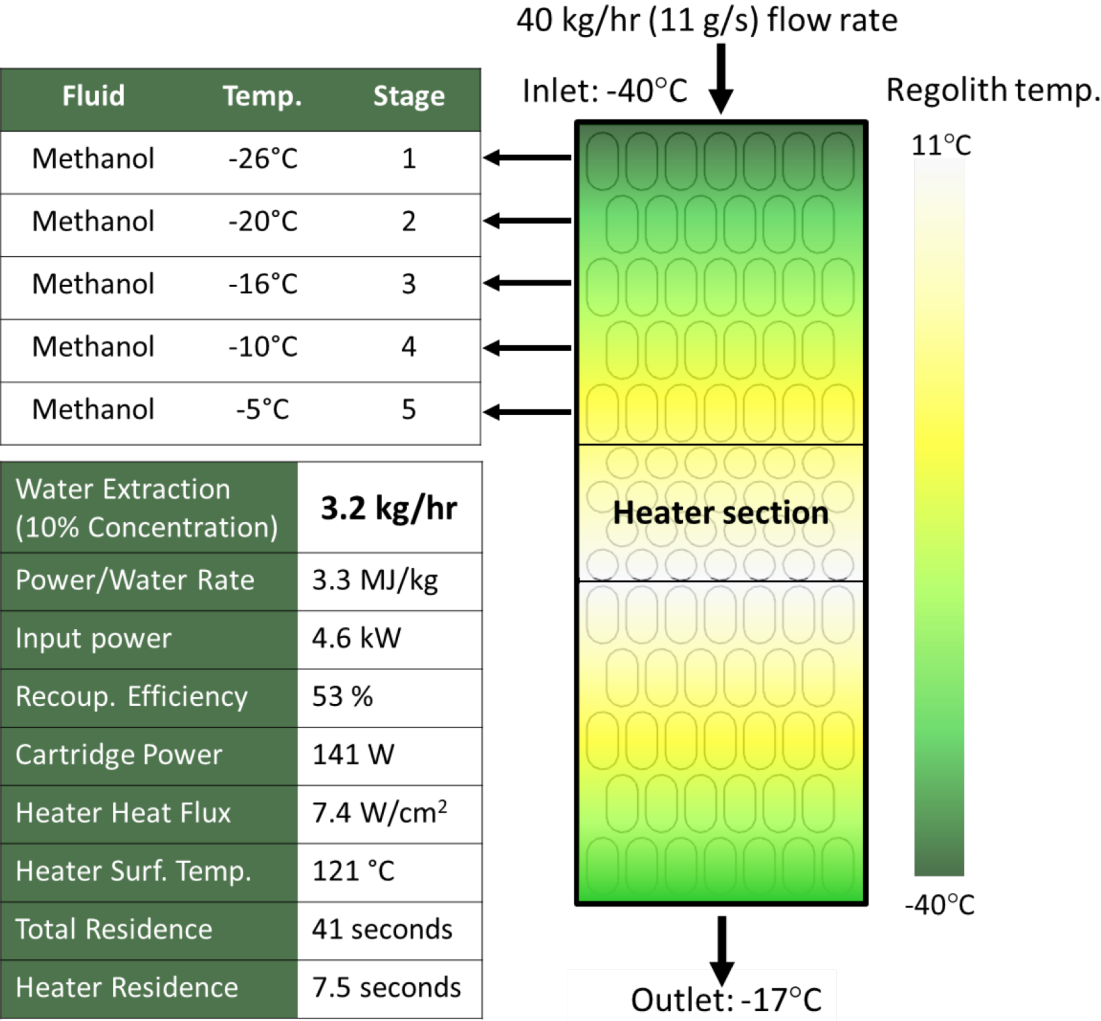
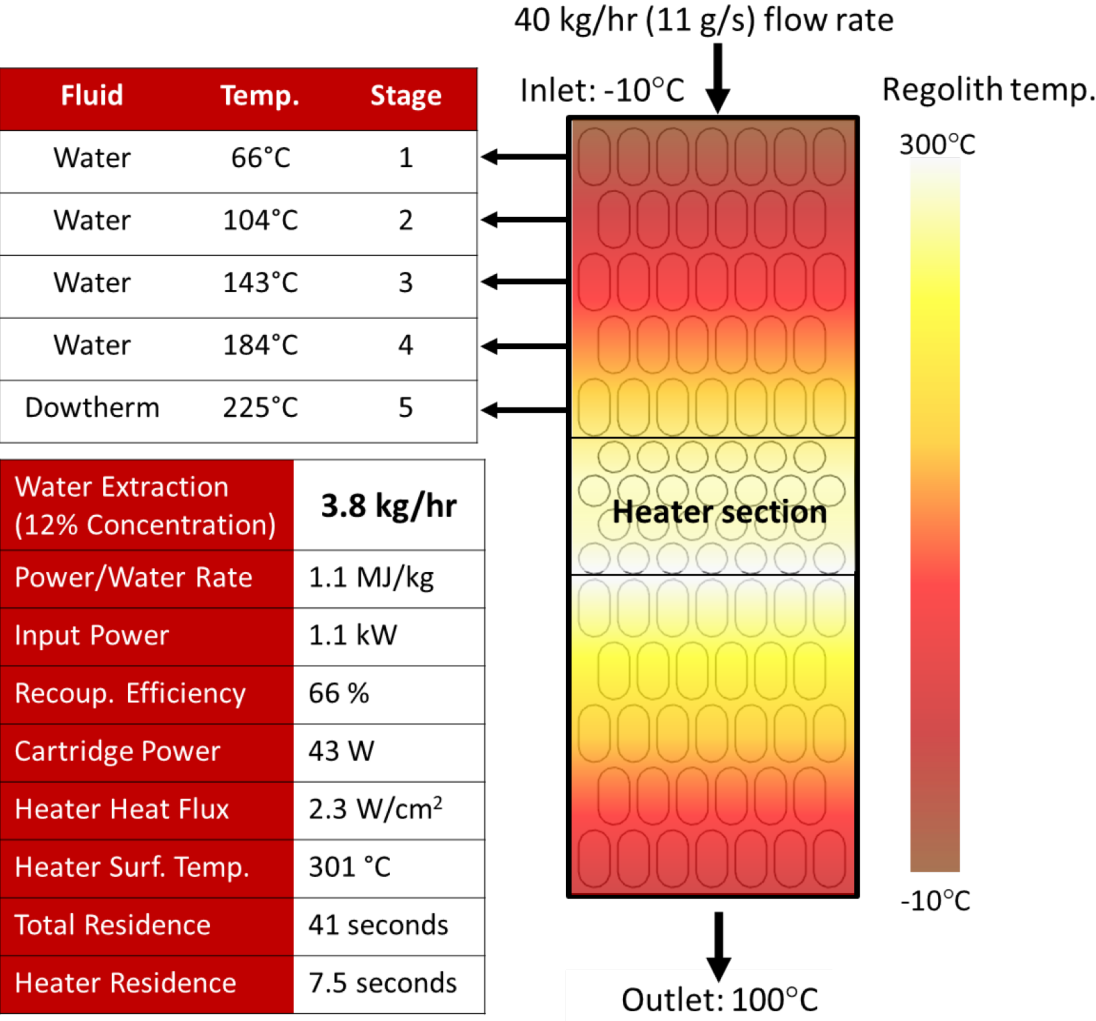


Summary

- Modeling approach developed to design heat pipe heat exchangers for regolith
- Modeling results will be compared against experimental results in the Helium Extraction and Acquisition Testbed (HEAT) device
- Demonstration of volatile extraction in a heat pipe heat exchanger before the end of this research effort
- The modeling approach for heat pipe heat exchangers could be useful for the extraction of water from hydrated and/or icy regolith



Modeling for Water Extraction on Mars (Ice or Hydrated)



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